

Corsair: UCI's Lightest 4WD Off-Road Vehicle

Powertrain Transmission Subteam

Advisors: John Michael McCarthy, Robert "Smitty" Smith, Phil Chipman, Ron Kessler, Jon Gribble Subteam Members: Nathan Martinez, Andy Hall, Jacob Montenegro, Matthew Shay, Samantha Lee



Table 1. Transmission Performance Curve

Engine Speed (RPM)	Engine Torque (ft-lb)	CVT Ratio	Rear Transfer Case Ratio	Front Diff Ratio	RWD Output Torque (ft-lb)	Speed (MPH)
2400	18.5	3.9	8.38	3.7	604.62	5.02
2600	18.1	3.9	8.38	3.7	591.54	5.44
2800	17.4	3.9	8.38	3.7	568.67	5.86
3000	16.6	3.9	8.38	3.7	568.67	6.28
3200	15.4	1	8.38	3.7	129.05	26.12
3400	14.5	1	8.38	3.7	121.51	27.76
3600	13.5	0.9	8.38	3.7	101.82	32.66

Table 2 & 3. Transmission Requirements

Requirement	2025 Design Target	Reason	
Peak Torque at Rear Wheels	604.58 ft-lbs	Calculated to complete traction event	
Top Speed	32.6 mph	Ensure sufficient torque and speed	
Overall Ratio (Rear)	32.68	Calculated to complete traction event	
Transfer Case Ratio	8.38	Calculated to complete traction event	
Transfer Case Weight	39.73 lbs (39.3% Decrease from 2024)	At least 30% Reduction (From 2024 transfer case + Differential)	
4x4 Type (Chain, Shaft, etc.)	Toggleable 2WD/4WD Driveshaft system		

Requirement	Scoundrel 2024	Corsair 2025	
Peak Torque at Rear Wheels	543.5 ft-lbs	604.58 ft-lbs (11.2% Increase)	
Transmission Weight	220.3 lbs	148.6 lbs (32.5% Reduction)	
Hill Climb Angle	45°	76°	

Transmission Breakdown

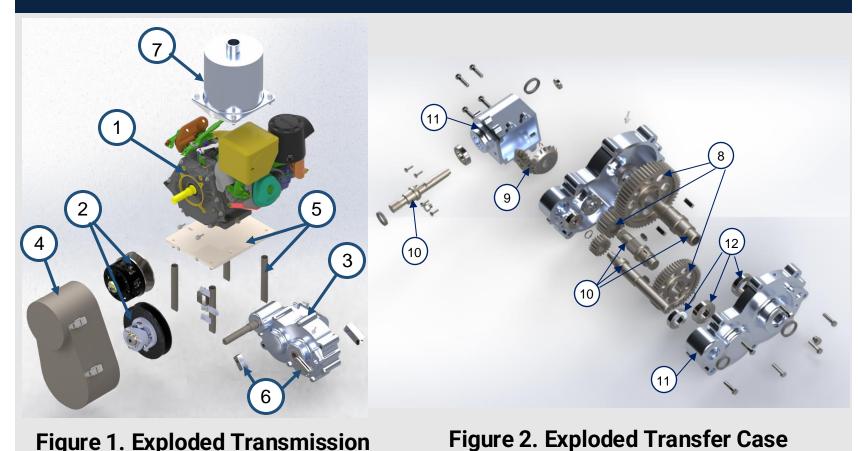


Figure 1. Exploded Transmission

Kohler CH440 Engine

Transfer Case

CVT Guarding

GX9 CVT Transmission

Bevel Gears 5 Engine Mounting Transfer Case Mounting Shafts

11

Figure 4. Transfer Case Gears

Housing

12 Bearings

Transf	or C	000	Doois	
Hallst	er u	ase	DESIG	П

Fuel Tank

Spur Gears



Figure 3. Transfer Case and Brake **Assembly Isometric View**

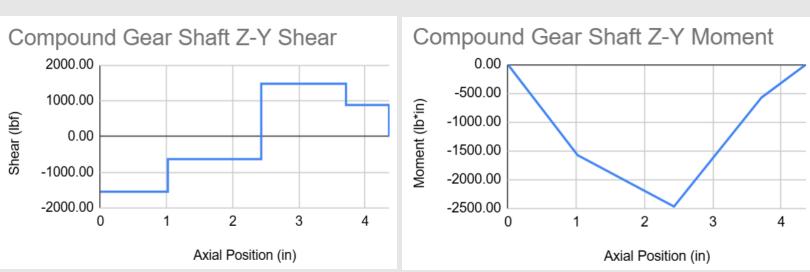
2000.00

1000.00

-1000.00

-2000.00

0.00



Transmission Design & Manufacturing

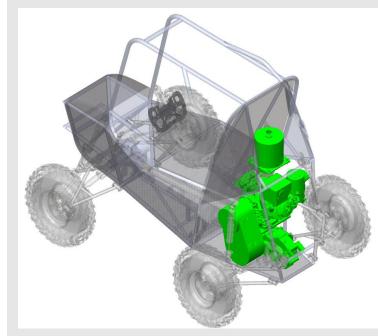


Figure 5. Transmission **Components Integrated into** Corsair



Figure 6. In-Progress **Gear and Shaft Machining With Local Sponsor**

Transfer Case FEA

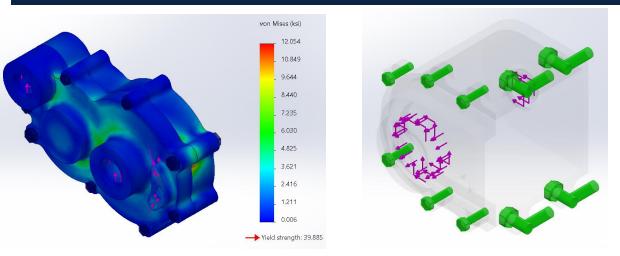
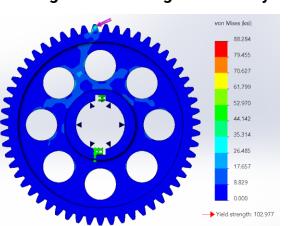


Figure 7. Housing Static Analysis

Figure 8. Housing Bolt Analysis



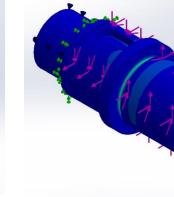
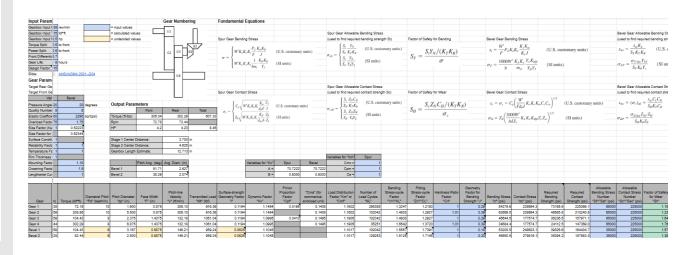


Figure 9. Gear 4 Webbing Analysis

Figure 10. Compound Shaft Static Analysis

AGMA 2001—D04 Gear Calculations





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Powertrain Driveline Subteam

UCISamueli School of Engineering Department of Mechanical and Aerospace Engineering

<u>Advisors:</u> John Michael McCarthy, Robert "Smitty" Smith, Phil Chipman, Ron Kessler, Jon Gribble <u>Subteam Members:</u> Sophia Catania, Soham Apte, Julian Prieto, Jason Dick, Michael Russell, Neal Purohit

Driveshaft Safety Margin Analysis

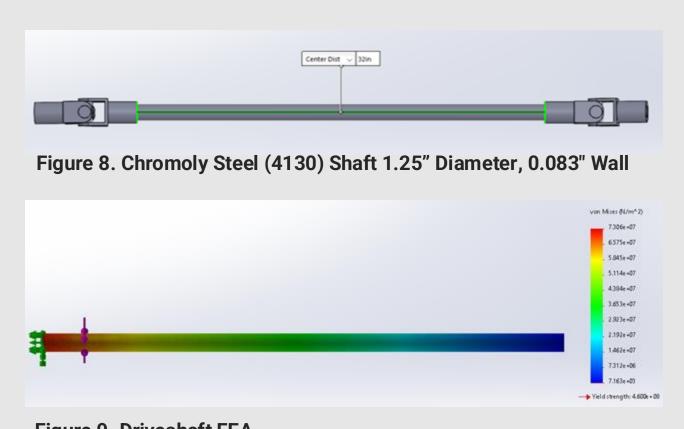


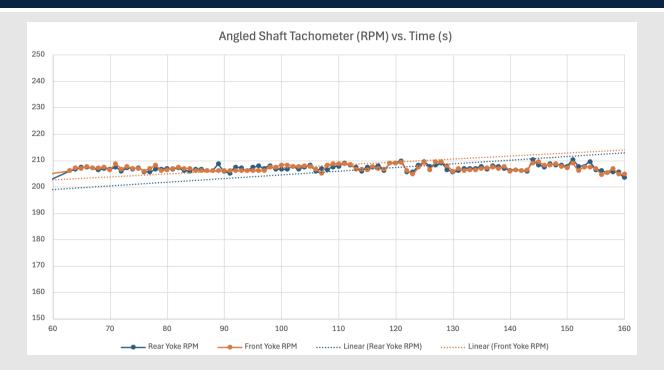
Figure 9. Driveshaft FEA

	Peak Transmission Output Value	Driveshaft Critical Value
Torque (ft-lb)	83.54	505.7
Critical Speed 1727.27 (RPM)		6395.2

Operating at only:

16.52% of critical torque and 27.02% of critical speed

U-Joint Phasing Verification



U-joint phasing verified by measuring and plotting the RPM of both output yokes simultaneously. The data confirms that both yokes rotate at consistent speeds, with minor outliers likely due to noise or transient effects.

Driveline Breakdown

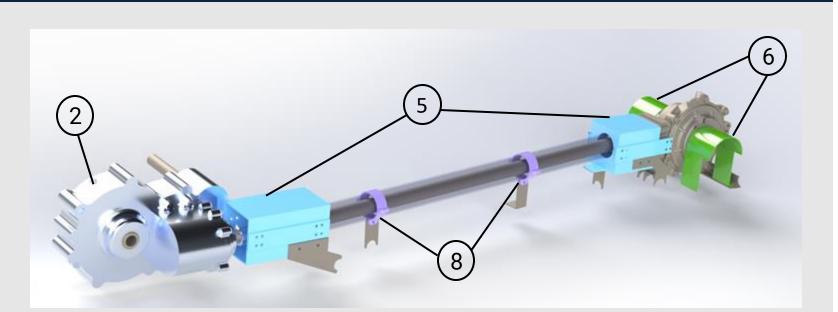


Figure 1. Driveline Assembly

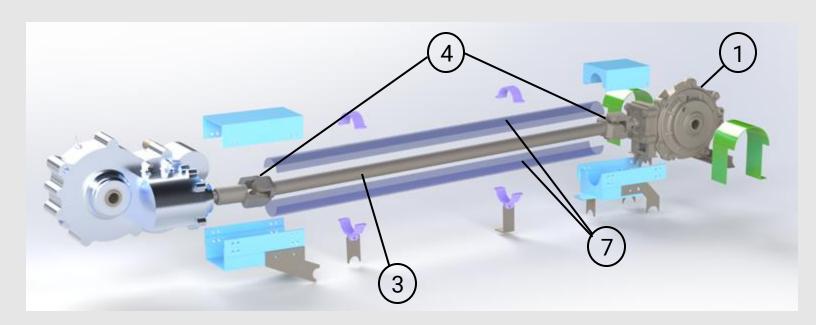


Figure 2. Driveline Exploded View

1	Polaris Front Differential	5	Yoke Guards
2	Gearbox	6	CV Axle Guards
3	Driveshaft	7	Driveshaft Finger Protection
4	U-Joints	8	Driveshaft Hoops

Next Steps

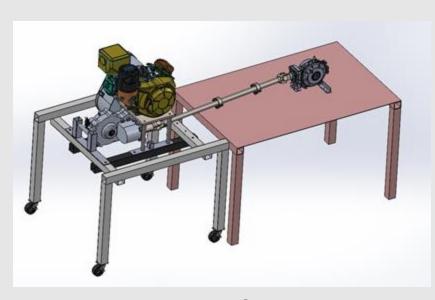


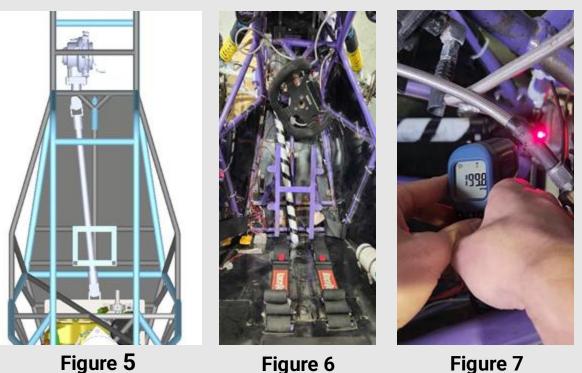
Figure 3. Test Stand Isometric View



Figure 4. Corsair Integration

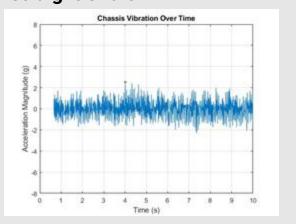
Scoundrel Testing

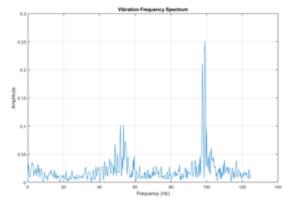




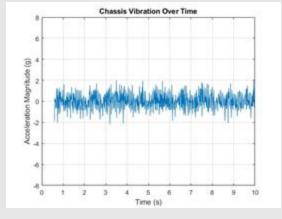
Driveline Vibrations: Impact of Driveshaft Angle

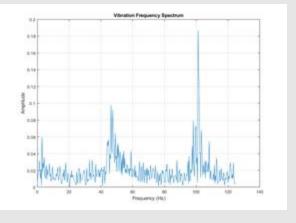
Straight Shaft





Angled Shaft (6 degrees)





Testing confirmed that the chassis did not experience additional vibrations or a new dominant frequency due to the driveshaft operating at a 6-degree angle, to accommodate the off-center gearbox output shaft.

Subteam Sponsor

driveshaftpro