Critical Design Review ZotSun: Front Suspension Subteam

Nicholas, Patrick, Quan, Sebastian, Tiffany



Overview

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This presentation shows and justifies our design decisions, and outlines our testing plans for them.





Parameters Being Tested

Parameter to Test	Requirement	Individual Responsible	R
Steering Ratio, Maximum Steering Angle	> 17 degrees	Nicholas	1(re
Steering Backlash	< 10 degrees at the steering wheel.	Nicholas	10 ba
Spring Rate	 > 250 lbf / in. (550 lbf / in. expected), Linear to within R² > 0.70, for the first half of suspension travel (1.2"). 	Quan	9. 10 22 10 21 20 20
Rocker & Rocker Mount Testing	Withstand 2g bump (330 lbf) at an angle of 8° (46 lbf lateral, 327 lbf vertical), at any rocker configuration.	Patrick	D in bı
A-Arm yield strength	Withstand 2g bump, 1g turn, 1g brake, and combined	Sebastian, Tiffany	D in bi

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Relevant Regulation

0.7.C: U-Turn equirement

0.7.D: Steering acklash

.3: All components > 00mm from ground, xcept wheels. 0.7.B: Wheel learance under full uspension ompression.

.1: "analysis shall nclude ... 1g turn, 2g ump, and 1g braking"

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Path of wheel

Illustration of the FSGP 10.7.C U-turn regulation, with our car. Drawn to scale.







Design Attributes

Attribute	Ο	C	F	M
Regulations: Compliance with racing and safety standards		X		
Minimize Weight: Essential for maximizing efficiency and performance	X			
Support Vehicle Load: Support the weight of the vehicle, driver, and subsystems			X	
Budget: Costs should be kept within the funding available for the project		X		
Test: Analysis of 3 unique loading conditions (1G turn, a 2G bump, and 1G braking)		X		
Material Selection: Parts will be manufactured out of 4130 alloy steel or OTS				X





Requirements

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- 10.7.D: Minimal Steering Backlash
- 10.7.C: U-Turn Radius Requirement
- 10.6.D: Non-Contact Parking Brake
- 10.9.B: Stability Under Crosswinds
- 10.5.C: Minimum Deceleration
- 10.5.B: Brake Pad Thickness
- 10.9.A: Figure-8 Course Completion
- 10.5.C: Brake Fade Resistance
- 10.8: Towing Requirements
- 10.2.A: Wheel Configuration Requirement
- 10.2.B: Tire Load Capacity(Dynamic)
- 10.2.D.5: Tire Load Capacity(Static)
- 10.2.D.6: Defect-Free, Speed-Rated Tires
- 9.3: Ride Height(>100 mm)
- 10.1.A: Shielding of Moving Parts
- 10.1.1: Suspension Clearance
- 10.4: Strength of Critical Fasteners
- 10.5.A: Dual Balanced Brake System
- 10.4.B: Locking Mechanisms for Safety Fasteners



Final Design



Collapse view of control arm and wheel mounting.



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Collapse view of steering rack mounting solution for prototype.



Final Design





Steering rack mounting plate; 0.125" carbon steel (A1008). Shape cut with CNC plasma cutting; drill holes to final size.

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Final Design



Old rack plate mount



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New rack plate mount





Compliance Table

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Technical Specifications	Compliant	Noncompliant	Not Tested Yet
1. Functional			
0 Degrees of Camber, +- 2 degrees	X		
12 degrees of KPI, +- 1 degree	X		
10.7.C: U-Turn Radius Requirement			X
2. Performance			
10.5.C: Minimum Deceleration of 4.72 m/s^2			X
10.5.B: Brake Pad Thickness Minimum of 6mm	x		
3.Physical			
10.2.B: Tire Load Capacity(Dynamic)			X
4. Safety			
10.1.1: Suspension Clearance	x (calculated)		



Final Key Decisions - Suspension Type: Double Wishbone

Ins Simulation MBD

odel name: Frons Study name: Bump(-P Plot type: Static dir



Front end top view

Scenario	A-arm	Vertical Load (lbs)	Longitudinal Load (lbs)	Lateral Load (lbs)
2-G Bump	Lower A-arm	316.8	0	0
2-G Bump	Upper A-arm	79.2	0	0
2.5-G Bump + Braking	Lower A-arm	456	158.4	0
2.5-G Bump + Braking	Upper A-arm	114	39.6	0
1-G Braking	Lower A-arm	218.4	158.4	0
1-G Braking	Upper A-arm	54.6	39.6	0
1-G Cornering	Lower A-arm	158.4	0	99
1-G Cornering	Upper A-arm	39.6	0	99



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Bump, brake, and cornering loads

URES (mm) 1.398e+00 1.258e+00 1.118e+00 9.794e-01 9.386e-01 5.591e-01 4.193e-01 2.795e-01 1.398e-02

Quan

Final Key Decisions - Shock Mounting



Figure: Racing aspirations calculator for vehicle suspension.



Simulation with Theoretical load of 550 lbs or 2G



$$WR = (0.86)^2 * 500 rac{lbf}{in} = 369.8 rac{lbf}{in}$$

 $F_s = -k_{eq}x$
 $-162lbf = 369.8 rac{lbf}{in}x$
 $0.44in = x$

- Chassis ride height with no load = 213mm; with vehicle weight, 202mm.
- With k_{eq} = 369 lbf / in = 64700 N/m, the natural frequency of the front is 4.7 Hz (very stiff).

Front Load Condition	Droop	Chassis Ride Height	Pass? (> 100m
0g (no load)	0mm	213mm	yes
1g (vehicle weight)	11mm	202mm	yes
2g (bump)	22mm	191mm	yes
≅ 2.45g (bump + brake)	27mm	186mm	yes



BOM, Manufacturing expectations, and final cost estimates



Purpose of Order:

Solar Car Front End Subsystem Prototype

VENDOR*	DESCRIPTION*	WEBSITE/LINK*	ITEM #/PART #* (if applicable)	QTY*	UNITS	UNIT PRICE*	EXTENDED PRICE	NOTES/LINK TO THE QUOTE in PDF
Aircraft Spruce	4130 STEEL TUBE	https://www.aircraftspru	03-08500-1	1		8.5	\$8.50	
Aircraft Spruce	4130 STEEL TUBE 3/4X.058	https://www.aircraftspr	03-04500-4	1		26.6	\$26.60	
Aircraft Spruce	4130 STEEL TUBE 5/8X.058	https://www.aircraftspr	03-03800-8	1		52	\$52.00	
Summit Racing	Summit Racing™	https://www.summitraci	SUM-420303	2		38.79	\$77.58	
Home Depot	2x2 and 1x2 lumber	https://www.homedepot		1		15	\$15.00	
Home Depoit	wood screws	https://www.homedepot	11598	1		7.99	\$7.99	
McMaster Carr	rod-end (right)	https://www.mcmaster.		4		7.95	\$31.80	
McMaster Carr	rod-end (left)	https://www.mcmaster.		4		7.95	\$31.80	
McMaster Carr	tie rod	https://www.mcmaster.		2		19.4	\$38.80	
Other Comments or	Special Instructions:					SUBTOTAL	\$290.07	
							\$22.48	
						Shipping		
						TOTAL	\$312.55	





Technical Risk Analysis (FMEA)









Component	Function	Failure Mode	Failure Cause	Effect	Corrective Action	Severity R
Fork End	Transmit forces from rear axle to the trailing arm.	Fatigue	Small radius of axle (R =)	Rear end falls to ground, causing significant chassis damage.	Fork end will be CNC machined; solid 4340 steel	8
Rocker	Transmit pushrod force into the shock, with a motion ratio < 1.	Fracture	Shear load from misaligned push rod.	Front end falls to ground, causing significant chassis damage.	Lateral support between two plates of rocker.	8
Steering rack mounting plate	Holds steering rack in place.	Fatigue (at welds)	The cyclic lateral loading and unloading provided by steering rack.	Sudden loss of steering control.	The plate is bridged, so that if a weld to the chassis fails, steering is still possible.	10
A-Arm	Constrain wheel path during suspension travel	Fatigue (at weld nuts)	Cyclic lateral loads from bumps; cyclic longitudinal loads from braking.	Front end falls to ground, causing significant chassis damage.	Scrutineer recommends horizontal rod end configuration.	8

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Missing Resources and Concerns

Manufacturing	Guidance in Analysis and Simulation	Material Selection
3D Printing and laser cutting: - Access to FABWORKS	 Mentors/Instructors: Access to experienced simulation engineers for support 	Material Properties: - Access to materials properties database and published research papers
Machining Resources: - Access to the machine shop - Access to hydraulic press	Simulation Software: - Access to FEA software such as SolidWorks and ANSYS	Suppliers: - Direct access to local suppliers able to adhere to our material requirements
 Welding Resources: Access to welding facilities Welding equipment Assembly tools 	Computational Resources: - Access to hardware capable of running simulations and analysis	 Prototyping Materials: Cheaper materials able to test the proof of concept Simple to assemble
 Budget: Budget allocations for the manufacturing process 		Testing Materials: - Materials intended to validify design of prototype



Gantt Chart

SIMPLE GAILLI GHANL DV VEUBA42.CUIII

https://www.vertex42.com/Excellemplates	s/simple-gantt-chart.hth	ni				Jan	20, 2	025			Ja	n 27,	. 202	25			F	eb 3,	202	5				Feb	Feb 10,	Feb 10, 202	Feb 10, 2025	Feb 10, 2025
TASK	ASSIGNED TO	PROGRESS	START	END	20 21 M T	1 22 W	23 : T	24 :	25 26 S S	27 M	28 2 T	9 30 N T	0 31	1 S	2 S	3 M	4 T	5 6 W T	7 F	8		9	9 10 S M	9 10 11 12 S M T W	9 10 11 12 13 S M T W T	9 10 11 12 13 14 S M T W T F	9 10 11 12 13 14 1 S M T W T F	9 10 11 12 13 14 1 S M T W T F 1
Steering							1 - 1	1	1.1	1 - 1					1	1						1	11.	1	1	1-1-1-1-1-	1	
Redesign and Mount Steering Mount	Nicholas, Quan	100%	1/22/25	1/29/25																								
Reinforce Wooden Jig with L Brackets	Patrick, Tiffany	100%	1/27/25	1/27/25																								
Measure Steering Angle	Nicholas	0%	1/27/25	2/5/25																								
Measure Steering Backlash	Nicholas	0%	1/27/25	2/5/25																								
Control Arms																												
Repurpose Metal Chassis for Testing	Sebastian, Tiffany	10%	1/27/25	2/6/25																								
Weld Mounts for Control Arm	Sebastian, Tiffany	0%	2/8/25	2/10/25																								
Perform Lateral Load Testing	Sebastian, Tiffany	0%	2/10/25	2/13/25																								
												-	1								_							







Gantt Chart

Spring Rate				
Determine Spring Resistance Under Loa	d Quan	0%	2/6/25	2/11/25
Ensure Structural Integrity is Maintained	Quan	0%	2/12/25	2/16/25
Ensure Correct Alignment Under Load	Quan	0%	2/17/25	2/22/25
Determine Suspension Travel	Quan	0%	2/23/25	2/27/25
Mounting Tabs/Rocker				
Complete Rockers/Mounts	Gyan	0%	1/28/25	2/10/25
Ensure it's able to Withstand 2G bump	Patrick	0%	2/10/25	2/18/25
Chassis Integration				
Integrate Components with Chassis	Everyone	0%	2/18/25	3/14/25







